



(300 Length*100 Width*3 thickness mm)

. (W/m^2 1500 – 250)

. (90° , 60° , 45° , 30° , 0°)

, (7.16 – 1.63 mm) (a)

(16 – 2 Hz) (f)

. ($138.991 < Ra < 487.275$)

(60° 0°)

($\theta = 90^\circ$)

(13%)

(7.6 %)

ABSTRACT

The aim of this work is to perform an experimental study for the effect of forced vertical vibrations on free convection heat transfer coefficient, from flat plate made of Aluminum with dimension (300 Length*100 Width*3 thickness mm). It's heated under a constant heat flux of (250-1500 W/m^2) upward. The flat plate was located horizontally or inclined in multiple angles at range of (0° , 30° , 45° , 60° , 90°). The experimental study is carried out at a range of frequency (2-16 Hz), the amplitude at the range of (1.63-7.16 mm), and the Rayleigh number at the range ($138.991 < Ra < 487.275$).

The results of this study show that, the relation between the heat transfer coefficient and the amplitude of vibration is incrementally for inclination angles from (0° - 60°), and reaches a maximum ratio of (13%) in the horizontal position, except at the vertical position ($\theta=90^\circ$) the heat transfer coefficient decreases as the vibration response increases and the maximum decrease ratio occurs at (7.6 %).

m^2		A_s
m		a
m/s^2		$acc.$
Hz		f
m/s^2	(9.81)	g
$W/m^2.C$		h
$W/m^2.C$		h_v
$W/m^2.C$		$h_{av.}$
$W/m^2.C$		$hr_{av.}$
$Amp.$		I
$W/m.C$		k
$W/m.C$		k_p
m		L
–		Nu
m		P
–		Pr
W/m^2		Q
W		$Q_{conv.}$
W		$Q_{gen.}$
W		$Q_{rad.}$
–		Ra
–		Re_v
K		T_{air}
K		T_f
K		$T_{sur.}$
m/s		u_v
$Volt$		V
m		w
$1/K$		β

:

(Ra)

.(Nu)

Rodrig,) (2003)

[3] (et al

(315 mm height * 35 mm width)

) W/m²10000,5000,3000,2500,2000,1500,10
560) (00mm long * 470 mm height * 130 mm
(width. (10¹²)

:

.[1]

$$Nu = \frac{2}{360^{1/5}} \times \left[\frac{Pr}{\frac{4}{5} + Pr} \right]^{1/5} \times Ra_y^{1/5} \quad (1)$$

Fohanno &) (2005)

[4](Polidori

. [2]

$$Nu = 118.6350 * Ra_v^n \quad (8)$$

$$Nu = 118.6350 * Ra^{0.53506} * Re_v^{-0.11269} * \theta^{-1.30968} \quad (9)$$

) (2009)

[6] (

$$Nu_x = 0.496(Ra_x)^{1/5} \quad (2)$$

$$Nu_x = 0.073(Ra_x)^{2/7} \quad (3)$$

(16mm)
(48mm)

$$Ra_x > 6.3 \times 10^9$$

() (2007)

[5]

(45°, 0°)

(f)

2.2 - 0) (a)

(16-2 Hz)

(300mm)

(mm

(16mm)

(1500-500 W/m²)

(48mm)

(1500-500)W/m²

(13.34%)

(f)

(60°-0°)

1.79-) mm (a)

(20-2 Hz)

(0

(Re_v)

(Ra)

$$h = [(2 * 10^{-3}) - (2 * 10^{-5} * Re)] * \theta + 0.004 * Re + 3.98 \quad (10)$$



.(Teflon)

:

(U)

300mm)

(long * 100mm width * 3mm height

(30)

(1000 W) (V*I)

(1mm)

.(1)

(1.5mm)

(Varic)

.(super glue)

(K)

(30)

(0.3mm)

(0.06mm)

Selector)

(Mica)

(Switch

(0.5mm)

(Thermometer)

)

. (3.5 3

Random)

)

(Function Generator

(

(Digital Oscilloscope)

(Power Amplifier)

(Vibration Exciter)

Vibration

(18mm)

(330mm*125mm)

(Meter)

:(A_t)

(0.003 m²)

$$A_t = W * L \quad (15) \quad (3) \quad (2)$$

$$T_{sav} = (T_1 + T_2 + \dots + T_n) / N \quad (16)$$

(T_f)

$$T_f = \frac{T_{sav} + T_{air}}{2} \quad (17) \quad (Q_{gen})$$

()

(Q_{conv})

(Q_{rad})

$$Q_{gen} = Q_{conv} + Q_{rad} \quad (11)$$

$$\beta = 1 / (T_f + 273) \quad (18)$$

$$Q_{gen} = I * V \quad (12)$$

(11)

$$Q_{rad} = \sigma * \epsilon * S_{sur} * A_t * (T_{sav}^4 - T_{air}^4) \quad (13)$$

: ϵ

(0.04)

(15)

[7] (1)

: S_{sur}

(Dimensionless)

Parameter

(T_f)

$$h = \frac{Q_{conv}}{A_t * \Delta T} \quad (14)$$

Vibration Reynolds)

:
(225)
(1500,1000,750,500,250 W/m²)
(90°,60°,45°,30°,0°)

Rayleigh number) (number
(Nusselt number) (

$$Re_v = \frac{2 \cdot \pi \cdot f \cdot a \cdot \delta}{v} \quad (19)$$

$$Ra = \frac{\beta \cdot g \cdot (T_{sav} - T_{air}) \cdot \delta^3}{v^2} \cdot Pr \quad (20)$$

$$Nu = \frac{h \cdot \delta}{k_f} \quad (21)$$

(θ)

:
(5,4)
(Amp.)
(u_v)
(250W/m²)

[5]

$$Ra = \frac{\beta \cdot g \sin \theta \cdot (T_{sav} - T_{air}) \cdot \delta^3}{v^2} \cdot Pr \quad (22)$$

:(8)

$$\delta = \frac{W \cdot L}{2(W + L)} \quad (23)$$

:(90°) (0°)

$$\delta = \frac{W \cdot L \cos \theta}{2(W + L \cos \theta)} \quad (24)$$

6)
(Hz
16)
(Hz

$$\delta = L = 300 \text{ mm} \quad (25)$$

(Re_v)

(δ) تمثل الطول المميز للصفحة.

$$Pr = \frac{\mu \cdot Cp}{k_f} \quad (26)$$

:(h_{vav})

(Vibration Velocity)

(6)

$$u_v = a \times f \quad (27)$$

$$a = \frac{acc. \cdot \sqrt{2}}{(2\pi \cdot f)^2} \quad (28)$$

(16,10,6,2 Hz)

(6 Hz)
 (30°)
 (16 Hz)

:

:

$$Nu = C * Ra^m * Re_v^n * \theta^l$$

(29) (C , m , l) (Nu)

(DGA-V1)

:

(11,10,9,8,7)

%)

(2.5-1)

(16,10,6 ,2 Hz)

:

(29)

(2)

(60°,45°,30°,0°)

(1)

- ")
- (1981)
- [2] M.C.Charrier Mojtabi, Y.P.Razi, K.Maliwan and A.Mojtabi, "Heat transfer due to high frequency vibration ", Int. J. Heat and mass transfer, (2003). -1
- [3] Rodrigo G.L. , Jose' A.F. And Douglas M.R. "Natural convection of Vertical flat plates" from Internet, (2003). (13%) -2
- [4] S. Fohanno, G. Polidori "Modeling of natural convective heat transfer at an internal surface "from Internet, (2005) . (7.6 %) -3
- " [5]
- "
- " (2007) -4
- " [6]
- " -5
- " (2009)
- [7] K.L.Kumar, "Engineering Fluid Mechanics", 5th.Edition, Boston, 1990.
- [8] P.K.Nag, "Heat and Mass Transfer", Second Edition, Tata McGraw-Hill Publishing Company Limited, 2007. [1]

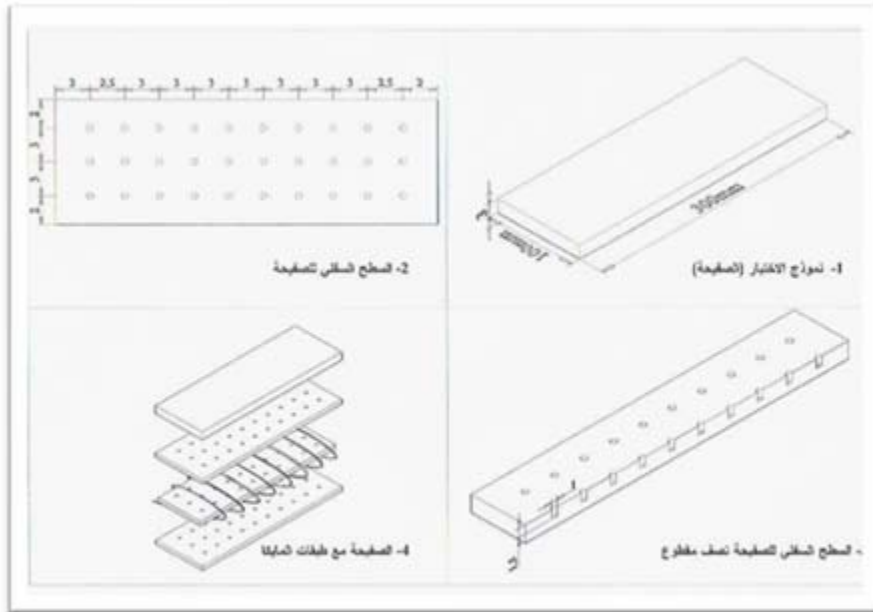
الجدول (1) يوضح النسب المئوية لقيم معامل انتقال الحرارة بوجود الاهتزاز إلى عدم وجوده.

θ	$f = 2 \text{ Hz}$	$f = 6 \text{ Hz}$	$f = 10 \text{ Hz}$	$f = 16 \text{ Hz}$
0°	12.0977	13.2894	11.3854	11.4888
30°	10.6678	4.0475	5.0203	5.386
45°	0.967	1.3721	1.0203	3.0976
60°	3.5358	2.5849	4.1174	5.3575
90°	7.1267	6.6189	7.6475	7.049

(29)

(2)

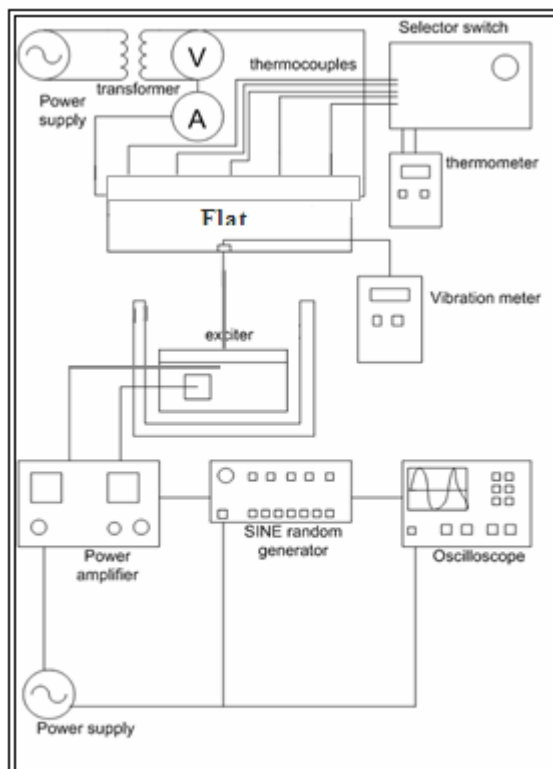
$f = 2 \text{ Hz}$				
Q (W/m^2)	C	m	n	l
250	1.433E-93	23.735	37.027	-0.8125
500	2.639	1.147	-0.693	6.92E-02
750	0.232	1.154	-0.703	-7.6325E-02
1000	11165.57	9.350E-02	-0.703	4.489E-02
1500	1.847E-12	4.53	4.42	-8.78E-02
$f = 6 \text{ Hz}$				
Q (W/m^2)	C	m	n	l
250	2.2E+71	-13.968	-27.448	0.556
500	4.75E-25	8.095	7.595	-0.1711
750	5.6E-02	1.841	-0.369	-0.0762
1000	527.155	0.746	-1.8155	5.409
1500	1.327E-13	4.457	4.5127	-7.9744
$f = 10 \text{ Hz}$				
Q (W/m^2)	C	m	n	l
250	573.7996	-0.000977	-0.2709	0.000329
500	4.799E-25	7.917	8.166	-0.146
750	0.447	1.587	-0.673	-7.235
1000	53.23	0.891	-1.2522	1.035
1500	0.0338	1.187	0.7966	0.1559
$f = 16 \text{ Hz}$				
Q (W/m^2)	C	m	n	l
250	8.65E-04	2.168	4.516	-6.485
500	3.127E-54	15.782	18.337	-0.939
750	0.4929	1.546	-0.601	-0.072
1000	0.276	1.67	-0.704	-1.667
1500	1.095E-11	4.044	3.75	-0.12155



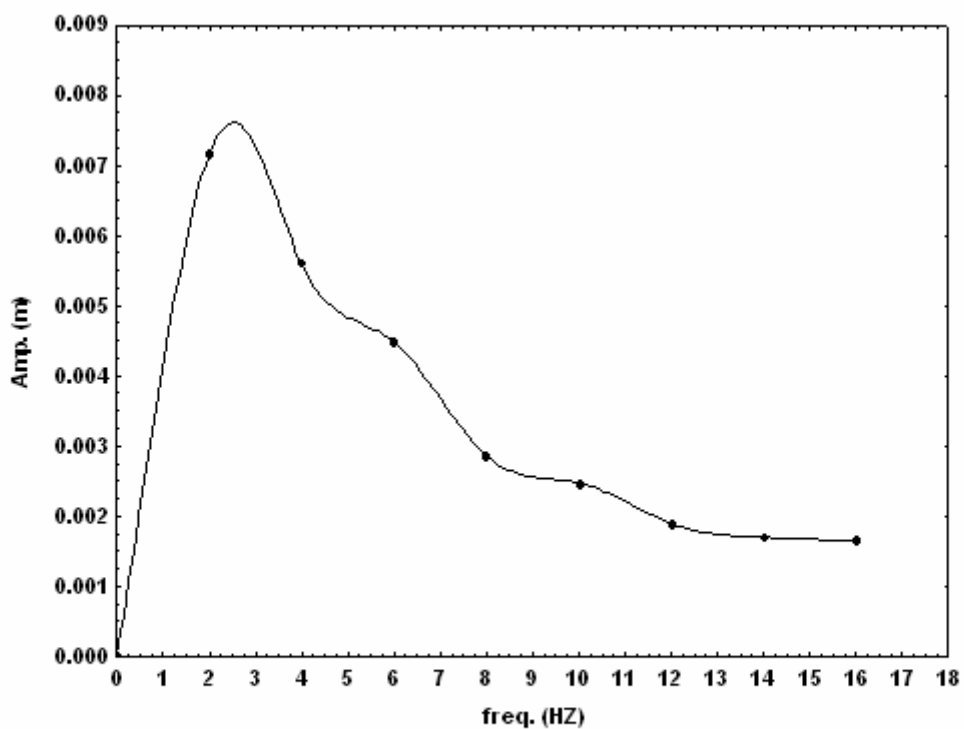
شكل (1) مخطط يوضح نموذج الاختيار المستخدم في البحث



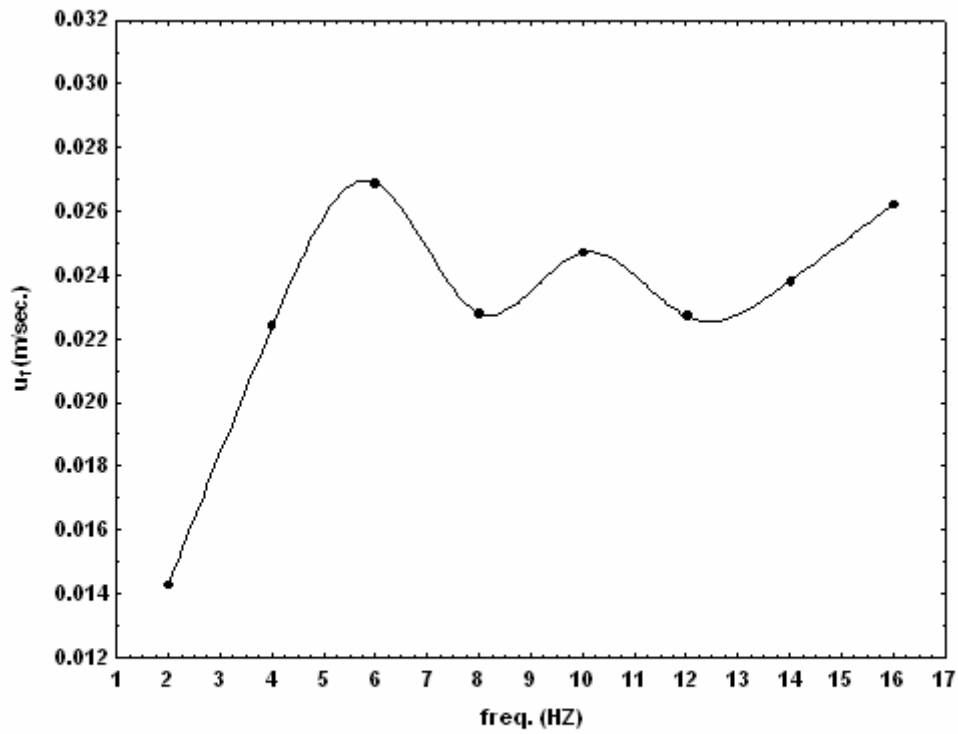
الشكل (2) صورة فوتوغرافية لمنظومة الاختبار



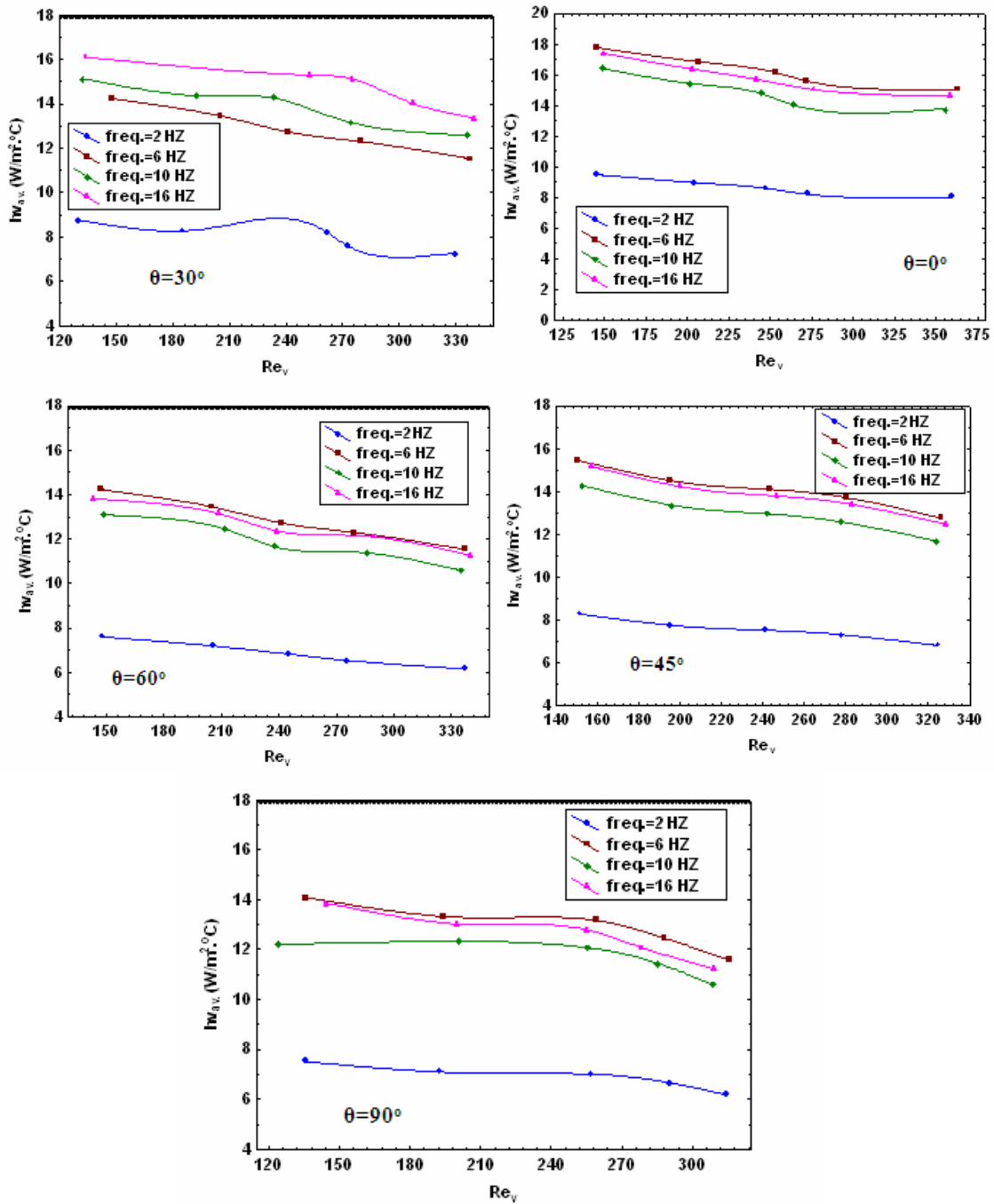
الشكل (3) يوضح منظومة الاختبار المستخدمة



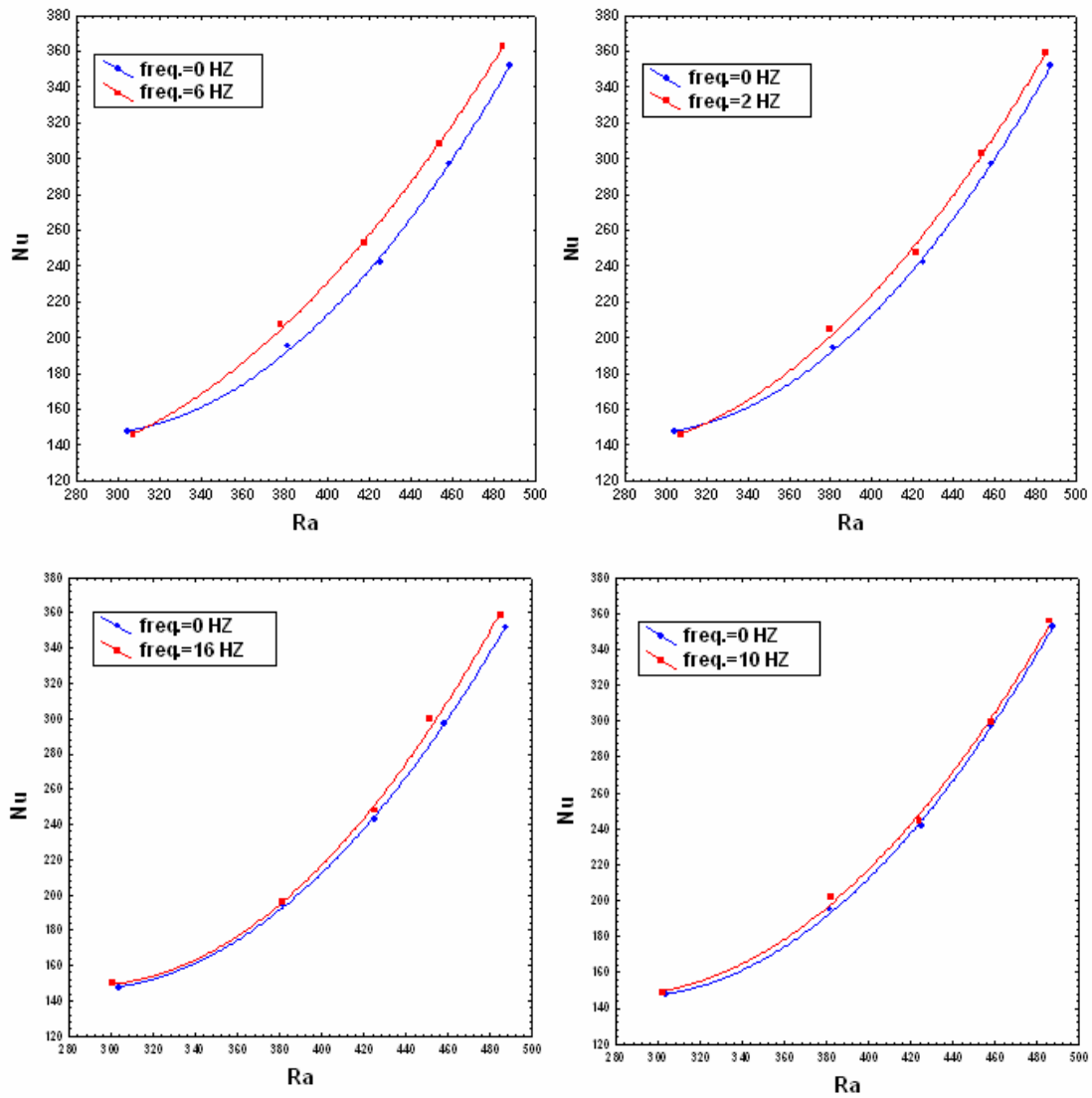
الشكل (4) يوضح تأثير تردد الاهتزاز في السعة لفيض حراري (250 W/m^2) ولزاوية ($\theta=0^\circ$)



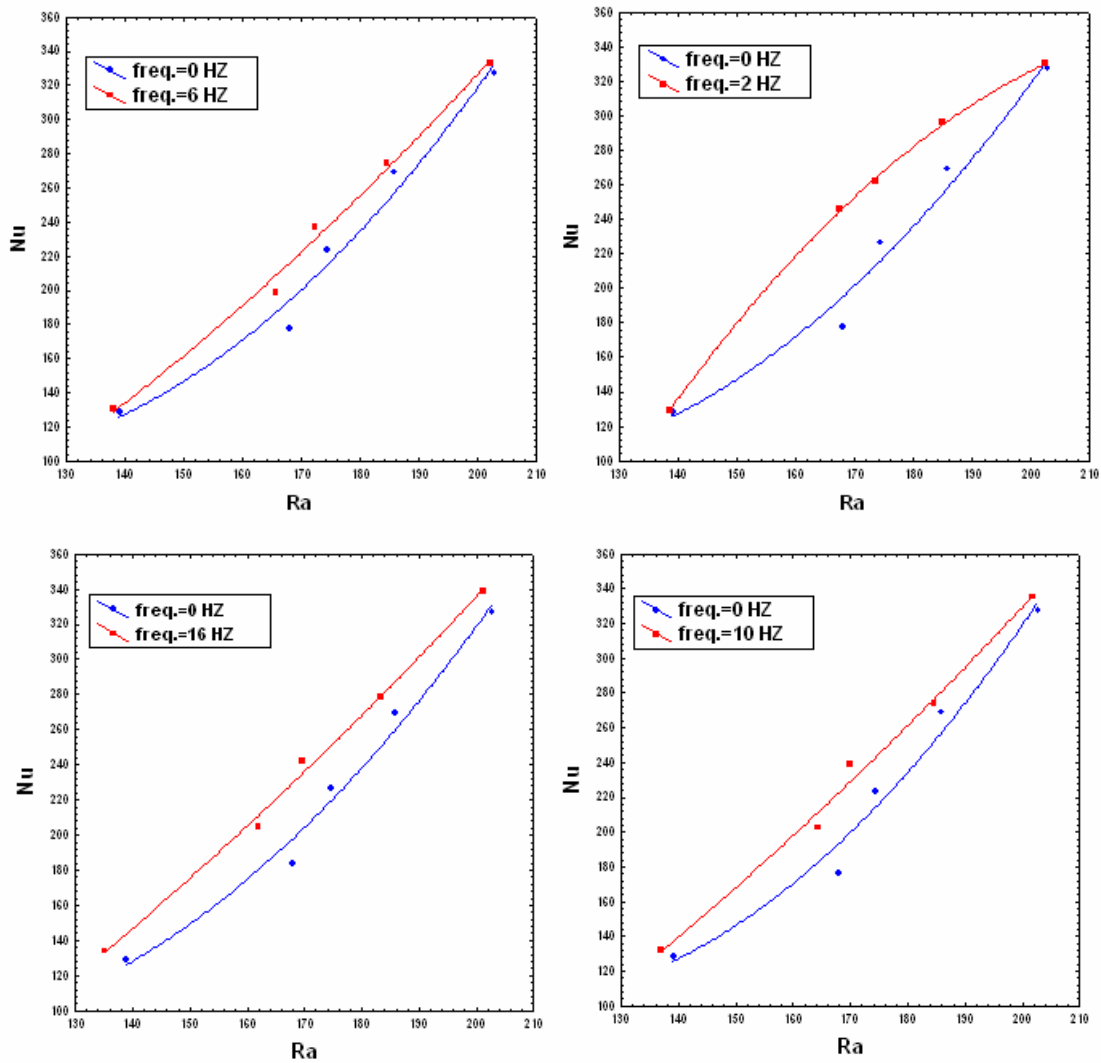
الشكل (5) يوضح تأثير تردد الاهتزاز في سرعة الاهتزاز لفيض حراري (250 W/m^2) ولزاوية ($\theta = 0^\circ$)



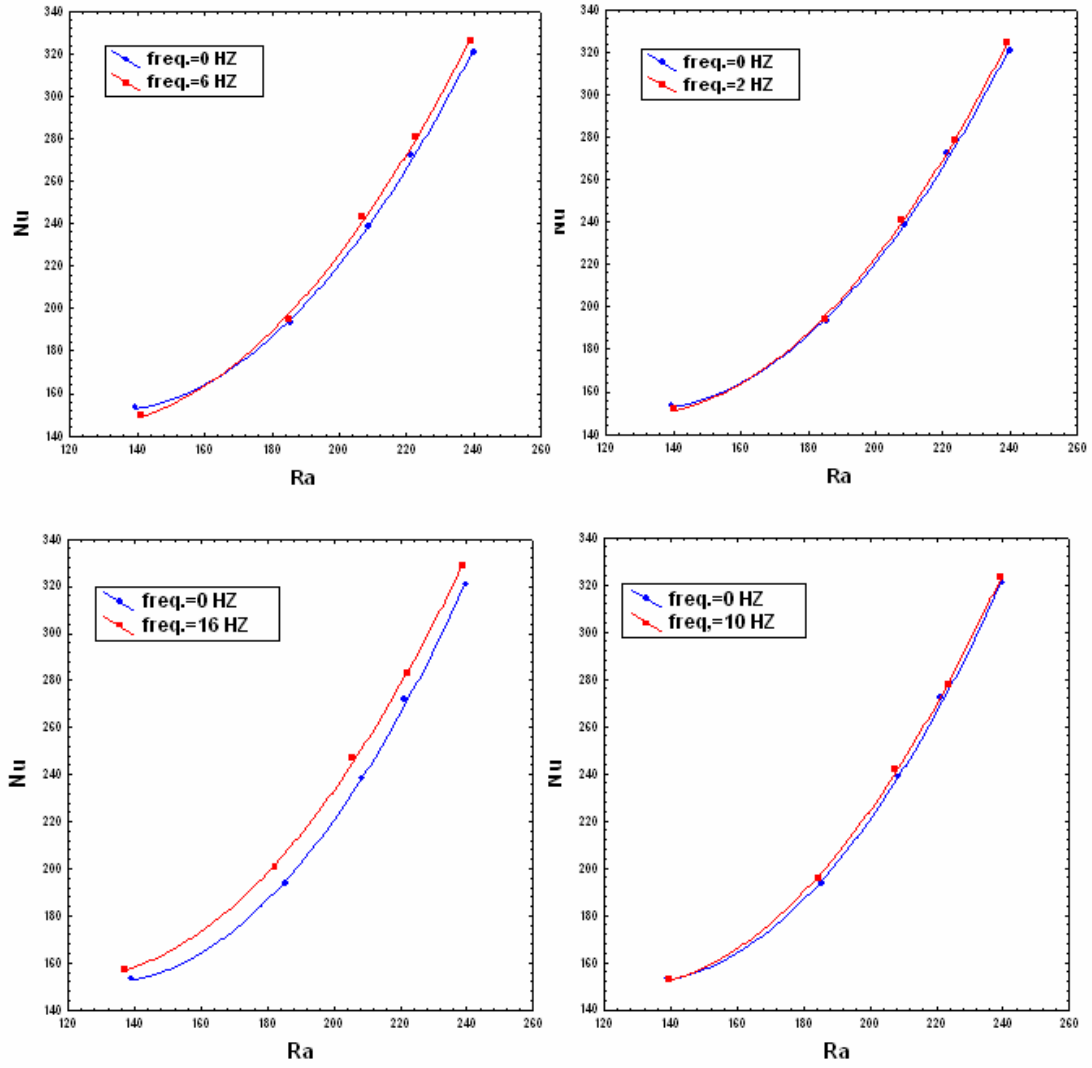
الشكل (6) يبين تأثير عدد رينولدز في معامل انتقال الحرارة الإجمالي الاهتزازي لمختلف مستويات الفيض الحراري



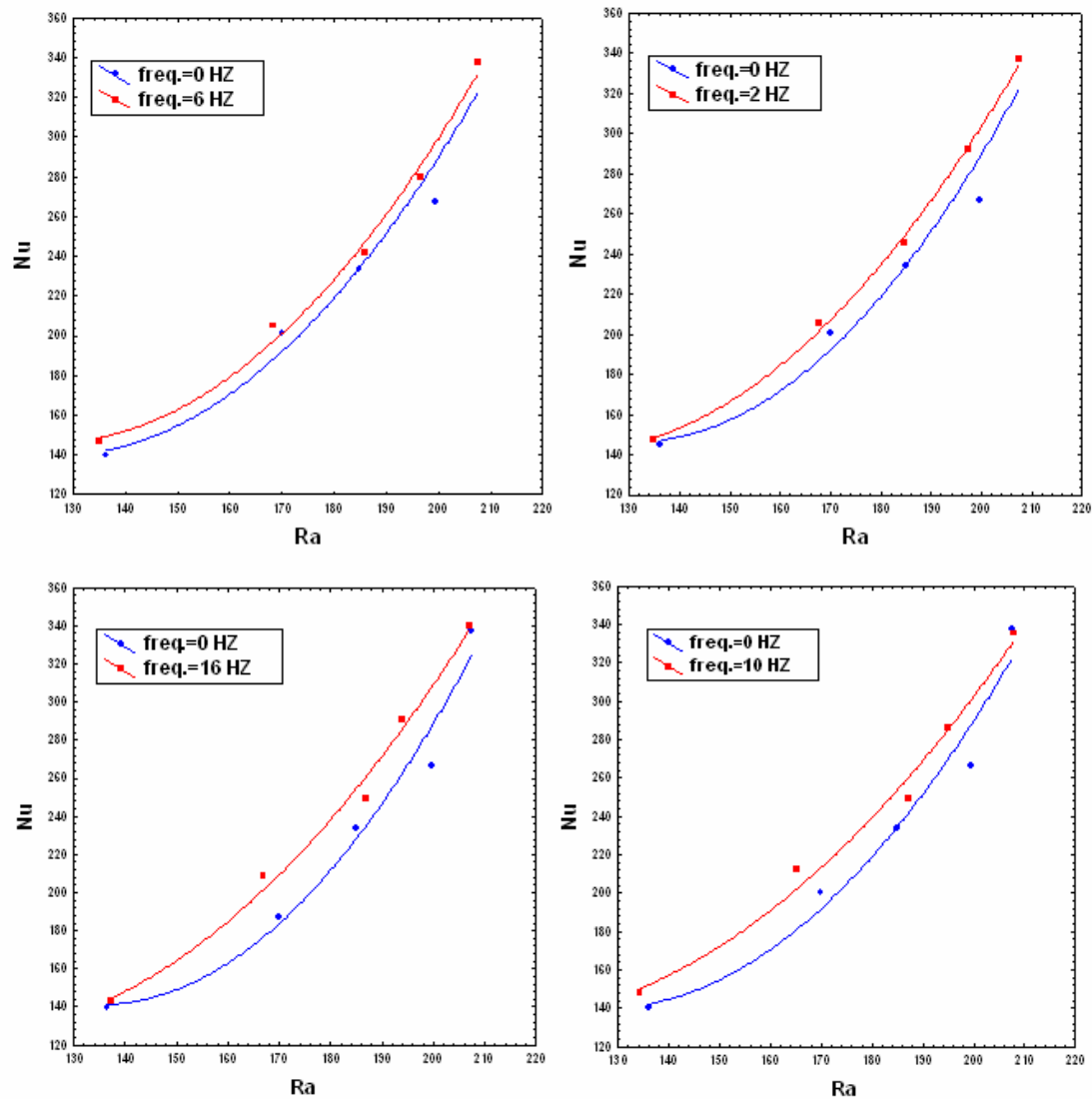
الشكل (7) يبين تأثير عدد رايلى في عدد نسلت الاجمالي بوجود وعدم وجود الاهتزاز للزاوية $\theta=0^\circ$ ولمختلف مستويات القيص الحراري



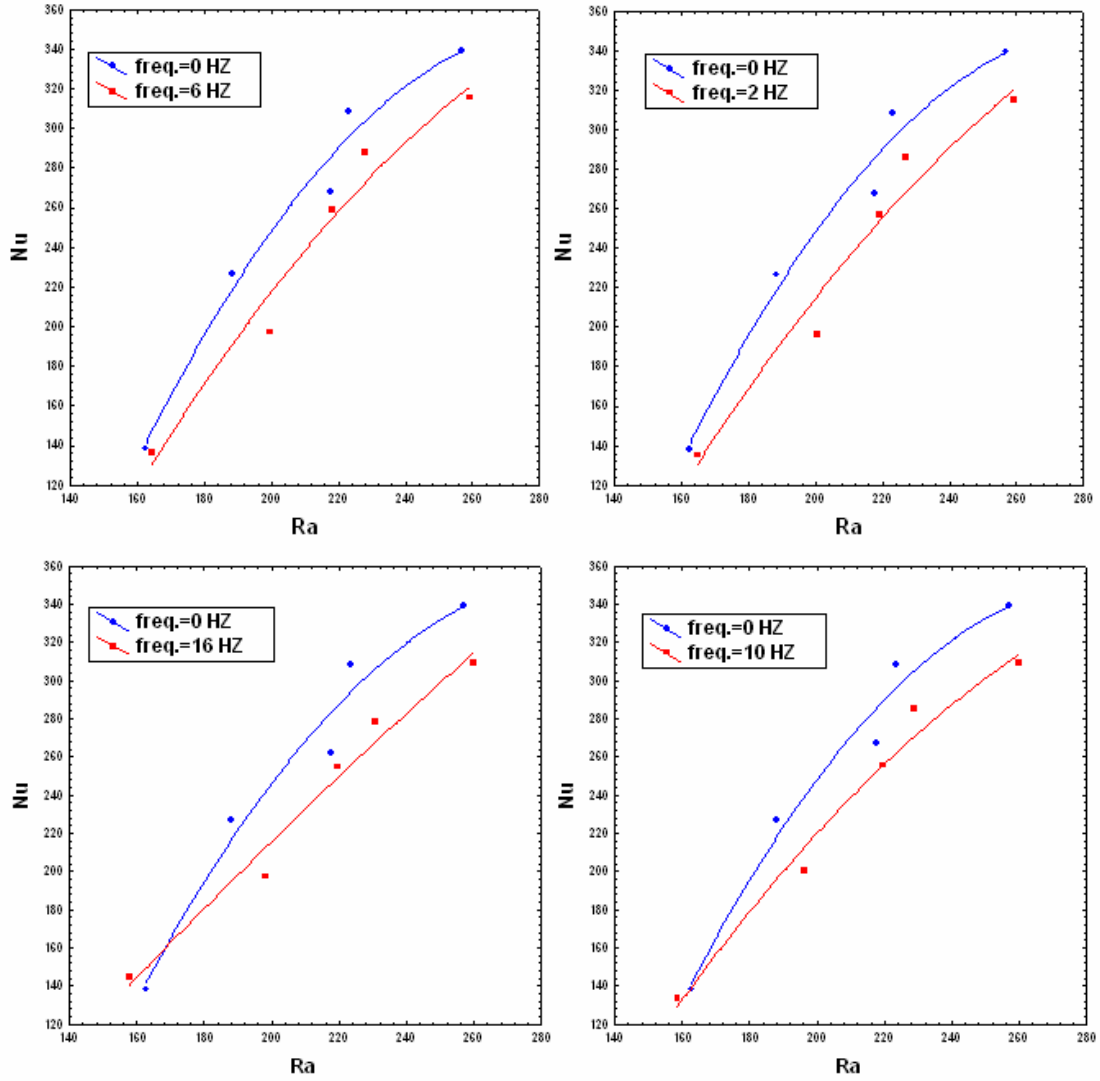
الشكل (8) يبين تأثير عدد راييلي في عدد نسلت الاجمالي بوجود وعدم وجود الاهتزاز للزاوية $\theta=30^\circ$ ولمختلف مستويات الفيض الحراري



الشكل (9) يبين تأثير عدد رابلي في عدد نسلت الإجمالي بوجود وعدم وجود الاهتزاز للزاوية $\theta=45^\circ$ ولمختلف مستويات الفيض الحراري



الشكل (10) يبين تأثير عدد رايلي في عدد نسلت الإجمالي بوجود وعدم وجود الاهتزاز للزاوية $\theta=60^\circ$ ولمختلف مستويات الفيض الحراري



الشكل (11) يبين تأثير عدد رايلى في عدد نسلت الإجمالي بوجود وعدم وجود الاهتزاز لتزاوية $\theta=90^\circ$ ولمختلف مستويات الفيض الحراري